### **RAPID CONSTITUENT PAYLOAD ENVIRONMENTS FOR MULTICONFIGURATION ORBITAL VEHICLES**

## quartus ENGINEERING FORWARD



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#### **OVERVIEW**

- Background:
	- Increasing demand for efficient dissemination of vibration load specifications
	- Vehicles with onboard payloads / critical subcomponents require additional testing & analysis on the *constituent* level
		- Interface environments typically recovered during pre-test analysis using explicit FEM simulation for all known bus-constituent configurations
	- For vehicles with a range of uncertain mission profiles and a high volume of flights, this explicit analysis can be extremely expensive and time consuming
- Quartus Engineering and Momentus Space have collaborated to develop a semi-automated Coupled Base Shake Analysis (CBSA) framework for efficient generation of constituent vibration environments
	- Specifically geared toward random vibration, but could be developed for sine environments as well
	- Uses simplified dynamic representation of constituents for rapid generation of database
	- Environments can represent a range of missions and constituent properties (mass, natural frequency, etc…) by simulating a wide variety of expected configurations
	- Results can be sorted by location, constituent mass, constituent dynamic properties
	- Test specifications defined as Acceleration Spectral Density (ASD) base input and appropriate force limits





#### CBSA PROCESS



### 1. MODELING APPROACH



### 1. MODELING APPROACH – BUS HCB REDUCTION

- Analysis performed in Nastran
- Bus FEM reduced in various configurations using Hurty Craig Bampton (HCB) reduction
- Interface grids included in CSET for attaching various payloads
	- Constituent interface grids attached to payload deck using RBE3's to avoid over-constraining deck motion at high frequencies









### 2. MODELING APPROACH – CONSTITUENT MODELS

- Tuned CBAR-CBUSH element models attached to bus HCB interface grids to represent dynamics of various constituents
- CBAR and CBUSH properties varied to populate expected mass/stiffness scatter

 $QP = 10-30$  kg

- Each constituent model consists of 6 CBAR segments joined with CBUSH elements
- CBUSHs required to decouple axial stiffness from modulus of bar elements

**Over 1300 configurations simulated with various constituent masses and locations as shown** 







#### **Constituent Dynamics Model**



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#### 2. RUN GENERATION



### 2. RUN GENERATION [COUPLED RUNS]

- Automatically generated 1300+ Nastran decks representing every combination of payload mass, stiffness, and location
	- Transfer functions recovered in each axis independently using Solution 111 (frequency response) in frequency range of interest
	- Included modal damping based on past vibration test correlation







### 2. RUN GENERATION [FIXED BASE CONSTITUENT RUNS]

- Also recovered fixed-base transfer functions of each constituent model
	- Fixed base results used to represent results from actual individual constituent tests
- Later applied ASD specification and force limits to simulate expected outcome of constituent tests







#### 3. APPLY NOTCHED BUS VIBE ENVIRONMENT



### 3. APPLY NOTCHED BUS VIBE ENVIRONMENT

- Applied bus interface random vibration environment to previously recovered transfer functions
- Automatically applied allowable notching to bus interface based on rideshare user's guide



#### **Example SpaceX Rideshare RV MPE + 3dB**

#### **Example Bus Interface Force Limiting (Axial Cases)**





- Fixed-base constituent test will never be able to reproduce in-axis coupled response due to incongruent interface impedance
- The coupled system interface (payload deck) may not have sufficient impedance to support & excite modal response of the constituent
	- Well known "vibration absorber effect" (VAE) documented by Scharton et al.
	- Observable as acceleration anti-resonances at coupled interface
	- Dependant on mass, inertia, mounting location, natural frequency of constituent
- Force limits should aim to introduce correct amount of load into constituent's primary structural mode

#### **Illustration of Transmissibility to Constituent Primary Resonance at Various Mounting Locations**





- The VAE is highly complex but the key property relevant to specification development is the expected transmissibility of force into the constituents primary structural mode
- RV Force limits can be algorithmically evaluated from CBSA as  $\mathcal{C}^2$ consistent with the Semi-Empirical Method (SEM):
	- Relationship between interface force and acceleration is frequency-dependent apparent mass function
	- Since environment spectrum and RMS are often driven by force and acceleration *peaks*, we are most interested in this apparent mass evaluated at the peak interface force & acceleration frequencies of the coupled system – evaluated as  $\mathcal{C}^2$  for random vibration
	- Appropriate  $\mathcal{C}^2$  will achieve correct interface force magnitude and simulate VAE antiresonance

#### **Standard Random Vibration SEM Force Limit (NASA STD 7004)**

$$
S_{FF}(f) = C^2 M_0^2 S_{AA}(f)
$$
,  $f < f_b$ 







#### FORCE LIMIT TRENDS

- Algorithmic  $C^2$  evaluation results in distribution of  $C^2$  values that can be analyzed statistically
	- Example of location trends shown below
- $C<sup>2</sup>$  trends by location, mass, and natural frequency can help characterize dynamics of the bus structure and helps inform...
	- Appropriate segregation of environments and associated  $C^2$  ranges
	- Placement of certain constituent payloads

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- Observation: Low  $\mathcal{C}^2$  does not always result in lower loading
	- High interface force may be due to ~quasi-static vibration associated with high acceleration levels









### 5A. EVALUATE CONSTITUENT ACCELERATION SPEC



#### INTERFACE SPECIFICATION – INDIVIDUAL CONSTITUENT

- Example ASD envelope and force limit implementation shown below for X-axis loading
	- In this example, the ASD specification is derived using industry standard approach (Spectral peak enveloping / FEMCI guidelines)
- For single constituent, force limit should limit force PSD to CBSA peaks and simulate VAE antiresonance



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#### **ENVELOPING METHODS:** SPECTRAL ENV. FOR LIMITED SAMPLE

- Below example environment customized for payloads on outer edges of the payload deck (lower  $C^2$  for X-axis loading)
	- $\mathcal{C}^2$  allowed to vary within range informed by previous step to match observed CBSA limit load
	- ASD level set to match RMS of P95/50 acceleration spectrum and follow spectrum shape
- Peak-enveloping approach may be used for constituent samples that have similar dynamics, but RMS acceleration will contain excess conservatism
	- ASD specification RMS will contain excessive conservatism due to peak-driven enveloping
	- Resulting interface forces may match limit load, but case-by-case assessment shows primary structure of all constituents were overtested



#### **ENVELOPING METHODS:** SPECTRAL ENV. FOR DIVERSE SAMPLE

- Below example environment includes all constituent locations, masses, and stiffnesses
	- $-c<sup>2</sup>$  allowed to vary within range informed by previous step to match observed CBSA limit load
	- ASD level set to match RMS of P95/50 acceleration spectrum and follow spectrum shape
- Peak enveloping approach for diverse sample of constituents will result in excessive conservatism in primary structure
	- Requires unrealistic reduction of  $C^2$  to achieve correct interface loads
	- Expect constituent primary structure to be significantly overtested



- Both acceleration specification and resulting interface forces will be excessively conservative for primary structure
- Reducing C<sup>^2</sup> further to match interface force would be unrealistic based on acceptable range evaluated in previous step



#### **ENVELOPING METHODS:** RMS ENV. FOR DIVERSE SAMPLE

- Below example environment includes all constituent locations, masses, and stiffnesses
	- $\mathcal{C}^2$  allowed to vary within range informed by previous step to match observed CBSA limit load
	- ASD level set to achieve desired Probability of Undertest (PoU) based on overall RMS
- RMS driven enveloping approach for diverse sample will maintain appropriate conservatism in primary structure, but introduces risk of undertesting small subcomponents (e.g. electronics)
	- Consider enforcing min workmanship environment floor or enveloping spectral peaks at high frequencies if this is of significant concern to the program



#### VALIDATION CASE EXAMPLE [GENERIC]

- A limited sample of explicit FEM results should be compared with CBSA sample to build confidence in modeling assumptions and desired level of conservatism
- In this example…
	- Explicit fixed-base simulation successfully enveloped explicit CBSA RMS force, moment, and acceleration
	- Element-by-element stress field comparison shows that acceptable fraction of constituent was undertested (<5%)



#### **Explicit FEM of True Mission Configuration vs Generalized CBSA Sample**

**Fraction of** *Structure Overtested*

#### **OBSERVATIONS**

- Coupled Base Shake Analysis (CBSA) can be performed to…
	- Generate random vibration environments and force limits for spacecraft across diverse range of configurations and constituent properties
	- Generate random vibration environments and force limits for spacecraft customized for specific mission configurations
	- Provide useful metrics for segregating environments (e.g. mass trends, impedance trends, etc…)
	- Aid in characterizing useful structural dynamics of bus structure
- A limited set of validation studies should be used to build confidence that…
	- CBSA produces results that bound desired output from explicit FEM analysis
	- CBSA produces
- When enveloping diverse CBSA samples, an RMS-driven enveloping approach may be desired to prevent excessive loads in primary structure
	- However, additional margin must be added depending on desired level of risk to ensure…
		- 1. High frequency enveloping of small subcomponent responses (e.g. electronics)
		- 2. Sufficient spatial coverage of envelope (i.e. account for incongruent load distribution fixed base vs coupled)
- Future work:

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- Quantify uncertainties and limitations associated with CBAR/CBUSH/RBE3 based CBSA approach
- Explore implications of varying modal mass distribution of constituent models
- Can CBSA provide indicators of how well spatial distribution of responses will match between coupled and fixed-base constituent results?
- Can CBAR internal VRS ensemble be used to build confidence in enveloping response of small subcomponents?
- Create automated process for constituent payload placement optimization to reduce environments



#### **REFERENCES**

- Soucy, Dharanipathi, Sedaghati, "Investigation of Force-Limited Vibration for Reduction of Overtesting"
- Scharton, "Force Limited Vibration Testing Monograph"



#### **BACKUP**



#### LATERAL IF SPECIFICATION EXAMPLE

• Included moment comparison for assessment of lateral environments

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- Observation: lateral loading associated with significantly higher force limits ervation: lateral loading associated<br>
significantly higher force limits<br>
Cross axis loading drives many force peaks (i.e.
	- lateral force driven by vertical motion)
	- In practice, vibe test can only generate force peaks in-axis motion





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#### CROSS AXIS RESPONSE CHECKS

- Current process allows for inspection of cross axis responses
- Off axis responses can be enveloped with in-axis responses



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#### VALIDATION CASE EXAMPLE [CUSTOMIZED]

- In this example...
	- A customized sub-sample of CBSA results was used to predict response of an explicit FEM
		- Subsample selected based on mounting location, mass properties, and natural frequency
	- Explicit FEA ASD and Force response fell within CBSA predictions, but some components on constituent structure were undertested
- Observation: Even if constituent reaches appropriate interface load, certain spatial regions of the structure may be undertested due to incongruent distribution of load

#### **Explicit FEM of True Mission Configuration vs Generalized CBSA Sample**



#### **Element-by-element Stress Field Error From Generalized CBSA Specification**





# PEAK ACCELERATION PSD VS C^2 CORRELATION –<br>AXIAL EXAMPLE

- Observed general trend of negative correlation between impedance (C^2) and interface ASD at frequency of peak force PSD
- Example below shown for all X-axis loading





- The VAE depends on 3 primary variables:
	- Mass & moment of inertia of constituent
	- Mounting location of constituent
	- Natural frequency of constituent
- Note: if the vibration absorber effect is present, it does not mean that the interface load will be low!
	- High quasi-static loading can come from supporting structure





